

SULIT



**KEMENTERIAN PENDIDIKAN TINGGI
JABATAN PENDIDIKAN POLITEKNIK DAN KOLEJ KOMUNITI**

**BAHAGIAN PEPERIKSAAN DAN PENILAIAN
JABATAN PENDIDIKAN POLITEKNIK DAN KOLEJ KOMUNITI
KEMENTERIAN PENDIDIKAN TINGGI**

JABATAN KEJURUTERAAN MEKANIKAL

PEPERIKSAAN AKHIR

SESI I : 2023/2024

DJJ20063: THERMODYNAMICS

TARIKH : 28 DISEMBER 2023

MASA : 8.30 AM – 10.30 PM (2 JAM)

Kertas ini mengandungi **TUJUH (7)** halaman bercetak.

Struktur (4 soalan)

Dokumen sokongan yang disertakan : Formula, Buku Stim

JANGAN BUKA KERTAS SOALAN INI SEHINGGA DIARAHKAN

(CLO yang tertera hanya sebagai rujukan)

SULIT

INSTRUCTION:

This section consists of **FOUR (4)** structured questions. Answer **ALL** questions.

ARAHAN:

*Bahagian ini mengandungi **EMPAT (4)** soalan berstruktur. Jawab **SEMUA** soalan.*

QUESTION 1**SOALAN 1**

- CLO1 (a) Aided with a suitable diagram, define:
Dengan bantuan gambar rajah yang sesuai, takrifkan:
- | | |
|-------------------|------------|
| i. System | [2 marks] |
| <i>System</i> | [2 markah] |
| ii. Boundary | [2 marks] |
| <i>Sempadan</i> | [2 markah] |
| iii. Surrounding | [2 marks] |
| <i>Sekeliling</i> | [2 markah] |
- CLO2 (b) Steam at 24 bar has the specific internal energy of 2350 kJ/kg. Relate the values given for solution of :
Stim pada tekanan 24 bar mempunyai tenaga dalam tentu 2350 kJ/kg. Hubungkaitkan nilai yang diberi untuk penyelesaian :
- | | |
|---------------------------|------------|
| i. dryness fraction | [4 marks] |
| <i>pecahan kekeringan</i> | [4 markah] |
| ii. specific volume | [3 marks] |
| <i>isipadu tentu</i> | [3 markah] |
- CLO2 (c) Steam at the pressure of 8 MN/m² has a specific volume of 0.03751 m³/kg. Referring to the steam table, calculate the value of specific internal energy of steam.

Stim pada tekanan 8 MN/m^2 mempunyai isipadu tentu $0.03751 \text{ m}^3/\text{kg}$. Dengan merujuk Jadual Stim, kirakan nilai bagi tenaga dalam tentu stim.

[12 marks]

[12 markah]

QUESTION 2

SOALAN 2

- CLO2 (a) 'Non-flow process is a process where no mass transfers across the boundary of a system'.

Based on the definition stated, give **FOUR (4)** examples of non-flow process.

'Proses tidak alir ialah proses di mana tiada jisim merentas sempadan sistem.'

*Berdasarkan takrifan yang dinyatakan, berikan **EMPAT (4)** contoh proses tidak alir*

[4 marks]

[4 markah]

- CLO2 (b) Table Q2 (b) indicated some properties value of helium which filled in a tank with a constant volume. The helium is heated in a process by heat transfer from the surrounding.

Table Q2 (b) menunjukkan nilai ciri-ciri bagi gas helium yang telah diisi ke dalam sebuah tangki dengan isipadu tetap. Gas helium tersebut dipanaskan oleh pemindahan haba dari persekitaran.

Table Q2 (b)/Jadual Q2(b)

Property <i>Ciri-ciri</i>	Initial <i>Awal</i>	Final <i>Akhir</i>
Pressure <i>Tekanan</i>	135 kN/m ²	330 kN/m ²
Temperature <i>Suhu</i>	38 ^o C	T₂
Volume <i>Isipadu</i>	0.5 m ³	
Molecular weight <i>Berat molekul</i>	4 kg/kmol	

Assuming that the gas acts as a perfect gas, relate the values given for solution of following properties:

Dengan mengandaikan gas adalah gas sempurna, hubungkaitkan nilai yang diberi untuk penyelesaian bagi ciri-ciri yang berikut:

- | | | |
|------|--|--------------------------|
| i. | Specific gas constant
<i>Pemalar gas tentu</i> | [2 marks]
[2 markah] |
| ii. | Mass
<i>Jisim</i> | [3 markah]
[3 markah] |
| iii. | Final temperature (^o C)
<i>Suhu akhir (^oC)</i> | [3 marks]
[3 markah] |

CLO2

- (c) A mass of 0.25 kg of perfect gas at a temperature of 450^oC and a pressure of 0.85 bar is compressed adiabatically to 4.75 bar. The given specific heat at constant pressure is 1.67 kJ/kg.K and the specific gas constant is 0.19 kJ/kg.K. Calculate the following:

Satu gas sempurna berjisim 0.25 kg pada suhu 450°C dan tekanan 0.85 bar telah dimampatkan secara adiabatic ke tekanan 4.75 bar. Diberi haba tentu pada tekanan tetap ialah 1.67 kJ/kg.K dan pemalar gas spesifik ialah 0.19 kJ/kg.K . Kirakan yang berikut:

- | | | |
|------|---|------------|
| i. | Final temperature | [7 marks] |
| | <i>Suhu akhir</i> | [7 markah] |
| ii. | Work transfer from or to the system | [3 marks] |
| | <i>Kerja yang dipindahkan daripada atau kepada sistem</i> | [3 markah] |
| iii. | Change in internal energy | [3 marks] |
| | <i>Perubahan tenaga dalam</i> | [3 markah] |

QUESTION 3

SOALAN 3

- | | | | |
|------|-----|--|-------------------------|
| CLO2 | (a) | State FOUR (4) devices that use the principle of flow process
<i>Nyatakan EMPAT (4) peranti yang menggunakan prinsip proses alir</i> | [4 marks]
[4 markah] |
| CLO2 | (b) | Air enters a nozzle steadily at density and velocity of 2.25 kg/m^3 and 50 m/s . It leaves at density of 0.615 kg/m^3 and velocity of 150 m/s . Given the inlet area of the nozzle is 85 cm^2 . Relate the values given for solution of:
<i>Udara memasuki muncung dengan mantap pada ketumpatan dan halaju 2.25 kg/m^3 and 50 m/s. Ia keluar dengan ketumpatan 0.615 kg/m^3 dan halaju, 150 m/s. Diberi luas bahagian masukan muncung ialah 85 cm^2, hubungkaitkan nilai yang diberi untuk penyelesaian bagi:</i> | |
| | i. | Mass flow rate through a nozzle
<i>Kadar alir jisim melalui muncung</i> | [4 marks]
[4 markah] |

ii. Exit area of the nozzle (cm^2) [4 marks]

Luas bahagian keluaran muncung (cm^2) [4 markah]

CLO2 (c) A rotary air pump is required to deliver 650 kg of air per hour having properties as in Table Q3(c).

Pam udara jenis rotary diperlukan bagi menghantar 650 kg udara per jam mempunyai ciri-ciri seperti Jadual Q3(c).

Table Q3 (c)/Jadual Q3 (c)

Property <i>Ciri-ciri</i>	Inlet <i>Bahagian Masukan</i>	Exit <i>Bahagian Keluaran</i>
Specific enthalpy <i>entalpi tentu</i>	355 kJ/kg	630 kJ/kg
Velocity <i>halaju</i>	15 m/s	22 m/s

If the rate of heat loss from the pump is 6000 W, calculate:

Sekiranya kadar kehilangan haba daripada pam ialah 6000 W, kirakan:

i. The power required to drive the pump [9 marks]

Kuasa yang diperlukan untuk memacu pam tersebut [9 markah]

ii. Inlet area if specific volume at entrance is $4.5 \text{ m}^3/\text{kg}$.

Luas bahagian masukan sekiranya isipadu tentu bahagian masukan ialah $4.5 \text{ m}^3/\text{kg}$

[4 marks]

[4 markah]

QUESTION 4

SOALAN 4

CLO2 (a) List **FOUR (4)** characteristics of reverse heat engine. [4 marks]

*Senaraikan **EMPAT (4)** ciri-ciri kitar enjin haba balikan.* [4 markah]

- CLO2 (b) A Carnot heat engine operates with an efficiency of 44% and rejects heat to the atmosphere at the temperature of 38°C.
Relate the information given for solution of hot reservoir temperature value in °C unit.
Enjin haba Carnot beroperasi dengan kecekapan sebanyak 44% dan menyingkirkan haba kepada atmosfera pada suhu 38°C. Hubungkan maklumat yang diberi bagi penyelesaian suhu takungan panas dalam unit °C.
[6 marks]
[6 markah]
- CLO2 (c) A steam power plant operates between a boiler pressure of 50 bar and a condenser pressure of 0.75 bar. If steam enters the turbine with dry saturated condition, calculate for a Rankine cycle:
Sebuah penjana kuasa stim beroperasi diantara tekanan dandang 50 bar dan tekanan pemeluwap 0.75 bar. Sekiranya stim masuk ke dalam turbin pada tekanan tepu kering, kirakan untuk kitar Rankine:
- i. Feed pump work [3 marks]
Kerja pam suapan [3 markah]
 - ii. Rankine efficiency [8 marks]
Kecekapan kitar Rankine [8 markah]
 - iii. Specific steam consumption [4 marks]
Penggunaan stim tepu [4 markah]

SOALAN TAMAT

1. PROPERTIES OF PURE SUBSTANCE

Steam

$$v = xv_g \quad h = h_f + xh_{fg} \quad u = u_f + x(u_g - u_f) \quad s = s_f + xs_{fg}$$

Ideal Gas

$$PV = mRT \quad R = \frac{R_o}{M} \quad R = C_p - C_v \quad \gamma = \frac{C_p}{C_v}$$

2. FIRST LAW OF THERMODYNAMICS

$$\Sigma Q = \Sigma W \quad Q - W = U_2 - U_1$$

Flow Process

$$\dot{m} = \rho CA = \frac{CA}{V}$$

$$h = u + pv$$

$$h = C_p \Delta T$$

$$Q - W = \dot{m} \left[(h_2 - h_1) + \left(\frac{C_2^2 - C_1^2}{2} \right) + (Z_2 - Z_1)g \right]$$

Non-Flow Process

1. Isothermal Process ($PV = C$)

$$U_2 - U_1 = 0 \quad Q = W$$

$$W = P_1 V_1 \ln \left(\frac{V_2}{V_1} \right) \quad @ \quad W = P_1 V_1 \ln \left(\frac{P_1}{P_2} \right)$$

$$W = mRT \ln \left(\frac{V_2}{V_1} \right) \quad @ \quad W = mRT \ln \left(\frac{P_1}{P_2} \right)$$

2. Adiabatic Process ($PV^\gamma = C$)

$$U_2 - U_1 = mC_v(T_2 - T_1) \quad W = \frac{P_1 V_1 - P_2 V_2}{\gamma - 1} = \frac{mR(T_1 - T_2)}{\gamma - 1}$$

$$Q = 0 \quad \frac{T_2}{T_1} = \left(\frac{P_2}{P_1} \right)^{\frac{\gamma-1}{\gamma}} = \left(\frac{V_1}{V_2} \right)^{\gamma-1}$$

3. Polytropic Process ($PV^n = C$)

$$U_2 - U_1 = mC_v(T_2 - T_1) \quad W = \frac{P_1V_1 - P_2V_2}{n-1} = \frac{mR(T_1 - T_2)}{n-1}$$

$$Q = \frac{\gamma - n}{\gamma - 1} \times W \quad \frac{T_2}{T_1} = \left(\frac{P_2}{P_1}\right)^{\frac{n-1}{n}} = \left(\frac{V_1}{V_2}\right)^{n-1}$$

4. Isobaric Process

$$U_2 - U_1 = Q - W$$

$$W = P(V_2 - V_1) = mR(T_2 - T_1)$$

$$Q = mC_p(T_2 - T_1)$$

5. Isometric Process

$$U_2 - U_1 = Q$$

$$W = 0$$

$$Q = mC_v(T_2 - T_1)$$

3. SECOND LAW OF THERMODYNAMICS

$$W_{net} = Q_H - Q_L$$

Heat Engine

$$\eta_{th} = \frac{W_{net,out}}{Q_H} = 1 - \frac{Q_L}{Q_H}$$

Refrigerator

$$COP_{R,rev} = \frac{T_L}{T_H - T_L} = \frac{1}{T_H/T_L - 1}$$

Heat Pump

$$COP_{HP,rev} = \frac{T_H}{T_H - T_L} = \frac{1}{1 - T_L/T_H}$$

Power Cycle

$$\eta_{Rankine} = \frac{W_T - W_P}{Q_B} = \frac{(h_1 - h_2) - (h_4 - h_3)}{(h_1 - h_4)}$$

$$Work\ ratio = \frac{W_T - W_P}{W_T} = \frac{(h_1 - h_2) - (h_4 - h_3)}{(h_1 - h_2)}$$

$$s. s. c = \frac{3600}{W_T - W_P} = \frac{3600}{(h_1 - h_2) - (h_4 - h_3)}$$